

PCT

WORLD INTELLECTUAL PROPERTY ORGANIZATION
International Bureau



INTERNATIONAL APPLICATION PUBLISHED UNDER THE PATENT COOPERATION TREATY (PCT)

(51) International Patent Classification⁶ :

B66B 7/06, 11/08

A1

(11) International Publication Number:

WO 98/29327

(43) International Publication Date:

9 July 1998 (09.07.98)

(21) International Application Number: PCT/FI97/00824

(22) International Filing Date: 19 December 1997 (19.12.97)

(30) Priority Data:

965243 30 December 1996 (30.12.96) FI
965242 30 December 1996 (30.12.96) FI

(71) Applicant (for all designated States except US): KONE CORPORATION [FI/FI]; Munkkiniemen puistotie 25, FIN-00330 Helsinki (FI).

(72) Inventors; and

(75) Inventors/Applicants (for US only): AULANKO, Esko [FI/FI]; Käenkatu 6 C 33, FIN-04230 Kerava (FI). MÄKIMATTILA, Simo [FI/FI]; Jupperinmetsä 11 A, FIN-02730 Espoo (FI).

(74) Agent: KONE CORPORATION; Patent Dept., P.O. Box 677, FIN-05801 Hyvinkää (FI).

(81) Designated States: AL, AM, AT, AU, AZ, BA, BB, BG, BR, BY, CA, CH, CN, CU, CZ, DE, DK, EE, ES, FI, GB, GE, GH, HU, IL, IS, JP, KE, KG, KP, KR, KZ, LC, LK, LR, LS, LT, LU, LV, MD, MG, MK, MN, MW, MX, NO, NZ, PL, PT, RO, RU, SD, SE, SG, SI, SK, SL, TJ, TM, TR, TT, UA, UG, US, VZ, VN, YU, ZW, ARIPO patent (GH, GM, KE, LS, MW, SD, SZ, UG, ZW), Eurasian patent (AM, AZ, BY, KG, KZ, MD, RU, TJ, TM), European patent (AT, BE, CH, DE, DK, ES, FI, FR, GB, GR, IE, IT, LU, MC, NL, PT, SE), OAPI patent (BF, BJ, CF, CG, CI, CM, GA, GN, ML, MR, NE, SN, TD, TG).

Published

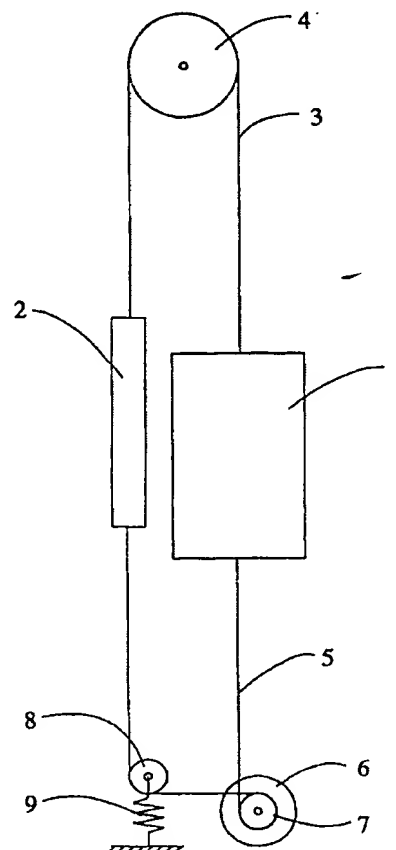
With international search report.

In English translation (filed in Finnish).

(54) Title: ELEVATOR ROPE ARRANGEMENT

(57) Abstract

Elevator rope arrangement in which the elevator car (1) and counterweight (2), travelling along guide rails in an elevator shaft, are supported by suspension ropes (3), which are attached to the top part of the elevator car (1) and passed via at least one diverting pulley (4) to the counterweight (2). Separate hoisting ropes (5) are attached to the lower part of the elevator car (1) and passed to lower part of the counterweight (2) via at least one diverting pulley (8). The hoisting rope (5) is a substantially thin rope made of synthetic fibre and having a sheath of plastic material.



FOR THE PURPOSES OF INFORMATION ONLY

Codes used to identify States party to the PCT on the front pages of pamphlets publishing international applications under the PCT.

AL	Albania	ES	Spain	LS	Lesotho	SI	Slovenia
AM	Armenia	FI	Finland	LT	Lithuania	SK	Slovakia
AT	Austria	FR	France	LU	Luxembourg	SN	Senegal
AU	Australia	GA	Gabon	LV	Latvia	SZ	Swaziland
AZ	Azerbaijan	GB	United Kingdom	MC	Monaco	TD	Chad
BA	Bosnia and Herzegovina	GE	Georgia	MD	Republic of Moldova	TG	Togo
BB	Barbados	GH	Ghana	MG	Madagascar	TJ	Tajikistan
BE	Belgium	GN	Guinea	MK	The former Yugoslav Republic of Macedonia	TM	Turkmenistan
BF	Burkina Faso	GR	Greece			TR	Turkey
BG	Bulgaria	HU	Hungary	ML	Mali	TT	Trinidad and Tobago
BJ	Benin	IE	Ireland	MN	Mongolia	UA	Ukraine
BR	Brazil	IL	Israel	MR	Mauritania	UG	Uganda
BY	Belarus	IS	Iceland	MW	Malawi	US	United States of America
CA	Canada	IT	Italy	MX	Mexico	UZ	Uzbekistan
CF	Central African Republic	JP	Japan	NE	Niger	VN	Viet Nam
CG	Congo	KE	Kenya	NL	Netherlands	YU	Yugoslavia
CH	Switzerland	KG	Kyrgyzstan	NO	Norway	ZW	Zimbabwe
CI	Côte d'Ivoire	KP	Democratic People's Republic of Korea	NZ	New Zealand		
CM	Cameroon			PL	Poland		
CN	China	KR	Republic of Korea	PT	Portugal		
CU	Cuba	KZ	Kazakstan	RO	Romania		
CZ	Czech Republic	LC	Saint Lucia	RU	Russian Federation		
DE	Germany	LI	Liechtenstein	SD	Sudan		
DK	Denmark	LK	Sri Lanka	SE	Sweden		
EE	Estonia	LR	Liberia	SG	Singapore		

ELEVATOR ROPE ARRANGEMENT

The present invention relates to an elevator rope arrangement as defined in the preamble of claim 1.

5

In traction sheave elevators, the elevator car and counterweight are suspended on round steel ropes. Normally, the same ropes act both as suspension ropes, whose function is to support the elevator car and counterweight, and as hoisting ropes serving to move the elevator car and counterweight. Therefore, the ropes must be designed to carry the entire load, even if, when a counterweight is used, the force needed to move the elevator is very small - in an extreme case nearly zero when the counterweight and the elevator car with the car load are equal in weight.

In prior art, there are also solutions having separate suspension ropes and hoisting ropes. Such an elevator is presented e.g. in US patent specification 5,398,781. In the elevator described in this specification, the suspension rope is attached to the top part of the elevator car and passed via diverting pulleys to a lever element on the counterweight. The hoisting rope is attached either to the top or bottom part of the elevator car and, like the suspension rope, passed via diverting pulleys and the traction sheave of the hoisting machine to a lever element on the counterweight. To compensate for rope elongation, the elevator described in this specification comprises a lever element fitted in conjunction with the counterweight and acting as a tensioning device. This patent focuses especially on the tensioning of the hoisting rope and contains no mention of any details of the suspension ropes or hoisting ropes. Neither does it describe any advantages that could be achieved

by separate implementation of hoisting ropes and suspension ropes.

The hoisting ropes generally used are steel cables,
5 whose friction coefficient is, however, so low that it has to be increased e.g. by using traction sheaves with different types of grooves or by increasing the angle of contact or angle of rotation of the rope around the traction sheave. In addition, a hoisting rope made of
10 steel functions as a kind of sound bridge between the hoisting motor drive and the elevator car, transmitting noise from the hoisting machinery to the elevator car and thus impairing passenger comfort.

15 A further drawback with prior-art solutions using steel hoisting ropes is that the bending radius of the rope is relatively large, which means that the traction sheave and diverting pulleys must have a large diameter. Another drawback with steel rope is that the weight of the
20 rope imposes a limit on the hoisting height of elevators. Moreover, steel ropes are liable to corrosion, so they require regular maintenance.

Specification EP 672 781 A1 presents a round elevator
25 suspension rope made of synthetic fibres. Topmost on the outside it has a sheath layer surrounding the outermost strand layer. The sheath layer is made of plastic, e.g. polyurethane. The strands are formed from aramid fibres. Each strand is treated with an impregnating agent to
30 protect the fibres. Placed between the outermost and the inner strand layers is an intermediate sheath to reduce friction. To achieve a nearly circular strand layer and to increase the volumetric efficiency, the gaps are filled with backfill strands. The function of the top-
35 most sheath layer is to ensure a coefficient of friction

of desired magnitude on the traction sheave and to protect the strands against mechanical and chemical damage and UV radiation. Thus, the load is supported exclusively by the strands. As compared with corresponding
5 steel rope, a rope formed from aramid fibres has a substantially larger load bearing capacity and a specific weight equal to only a fifth or a sixth of the specific weight of corresponding steel rope.

10 A drawback with these prior-art solutions, in which a round elevator rope formed e.g. from synthetic fibres, is that the rope has a relatively large bending radius, requiring the use of large-diameter traction sheaves and diverting pulleys. Further, there occurs a fair deal of
15 sliding of the strands and fibres in relation to each other. Moreover, the ratio of volume to area is high, which means that frictional heat will not be effectively removed from the rope and the rope temperature is therefore liable to rise unduly.

20

The object of the present invention is to eliminate the drawbacks of prior art and achieve a new type of elevator rope arrangement, in which the elevator ropes are
25 divided into two categories: a) suspension ropes, whose function is to connect the elevator car and the counterweight to each other and to support them, and b) a new type of hoisting rope made of synthetic material, whose function is to receive the unbalance between the counterweight on the one hand and the elevator car and its
30 load on the other hand and to move the elevator car.

In this arrangement, friction is not a necessary consideration regarding the suspension ropes, so these can be
35 made of steel cable. By contrast, the hoisting ropes are

thin ropes of synthetic material, in which the tensile strength of the structure is formed by longitudinal strands of e.g. aramid fibre. These strands are surrounded by a sheath that binds the strands of each rope together and provides a good friction coefficient against the traction sheave. The sheath is made of e.g. polyurethane, which gives a multifold friction coefficient as compared e.g. with steel rope. Details of the features characteristic of the solution of the invention are given in the claims presented below.

The hoisting ropes now only have to bear a fraction of the loads of the elevator, as they need not support the load resulting from the passengers or goods being transported and the counterweight. Therefore, the elevator hoisting rope of the invention can be made very thin, which means that it has a small bending diameter. The hoisting rope can also be implemented as a flat rope, in which case the sheath of the hoisting rope is of a planar shape and, in cross-section, the hoisting rope thus has a width substantially larger than its thickness.

The thin and flat hoisting rope allows the use of a traction sheave that is considerably smaller in diameter and lighter than those used at present. Therefore, also the moment required for moving the elevator car is low, and consequently it is possible to use a small and cheap hoisting motor. The flat band-like shape of the rope distributes the pressure imposed by the rope on the traction sheave or diverting pulley more uniformly on the surface of the traction sheave. Further, sliding of the fibres relative to each other is minimised, and so the internal shear forces in the rope are also minimised. In addition, the ratio of volume to area is low, which means that frictional heat is effectively trans-

mitted from the rope to the environment. Furthermore, the sheath of the hoisting rope can easily be coated with various materials, so the friction and abrasion characteristics can be optimised for different traction sheave materials. The small motor and small traction sheave are well applicable to an elevator without machine room because the hoisting motor with the traction sheave can be easily accommodated in the elevator shaft.

10 In the following, the invention will be described in detail by the aid of an example by referring to the attached drawings, in which

Fig. 1 presents an elevator rope arrangement according to the invention,

Fig. 2 presents another elevator rope arrangement according to the invention,

20 Fig. 3 presents a hoisting rope applicable to the elevator arrangement of the invention.

Fig. 4-8 present different synthetic-fibre rope solutions.

25

Fig. 1 shows a traction sheave elevator according to the invention, comprising an elevator car 1 and a counterweight 2 travelling along guide rails in an elevator shaft and suspended on suspension ropes 3. The steel suspension ropes 3 are fixed to the top part of the elevator car 1 and passed via a diverting pulley 4 in the elevator shaft to the counterweight 2. The substantially round hoisting ropes 5 used to move the elevator car and counterweight, made of synthetic material, are flexible and substantially thin as compared with the suspension

35

ropes. The hoisting ropes are attached by their first end to the lower part of the elevator car 1, from where the ropes are passed to the lower part of the counterweight 2 via the traction sheave 7 of a drive machine 6 placed on the bottom of the elevator shaft below the elevator car 1 and via a diverting pulley 8 placed on the bottom of the elevator shaft below the counterweight. The drive machine is e.g. a discoid electric motor of a flat construction in relation to its diameter, with a traction sheave integrated with the rotor and having a stator and rotor whose diameter is larger than the diameter of the traction sheave. The drive machine can be mounted either on the bottom of the shaft or on the shaft wall structures in the lower part of the elevator shaft. Several hoisting ropes running side by side can be used. In the solution illustrated by Fig. 1, the friction between the hoisting ropes and the traction sheave has been increased by having the hoisting ropes pass around the traction sheave 7 so that the hoisting ropes coming down from the elevator car pass between the diverting pulley 8 and the traction sheave 7 down to the traction sheave, run around the traction sheave by its lower side and then, having passed through a partial round about the traction sheave, go further by its upper side and intersect themselves, and after the intersection they go further to the diverting pulley 8, pass the diverting pulley by its lower side and go up to the counterweight. In this embodiment, the hoisting ropes are attached to the lower part of the counterweight.

30

In this suspension example, several thin hoisting ropes are used, but it is also possible to use a single flat rope. In the case of a flat rope, an additional difficulty results from the rope intersecting itself because the rope has a relatively large width. However, the rope

35

intersection can be implemented either by turning the traction sheave through an appropriate angle about its plane of rotation or by tilting the traction sheave in its plane of rotation. A further possibility is to both
5 turn the traction sheave and tilt it as described above, in which case the angle of turn or the angle of tilt will be smaller than when the traction sheave is only turned or only tilted. When separate hoisting ropes are used, the traction sheave also has to be tilted and/or
10 turned to allow the ropes to cross each other.

The hoisting ropes are tensioned between the elevator car and the counterweight by means of the diverting pulley 8. The tensioning is implemented using a tension
15 spring 9, which draws the traction sheave 8 so that the hoisting ropes always remain sufficiently tight on the traction sheave to provide the required friction regardless of elongation of the hoisting ropes. The tensioning can also be implemented using an arrangement in conjunction with the hoisting machinery, in which case the diverting pulley is fixedly mounted. In this case, the
20 mass of the hoisting machinery can be utilised for the tensioning of the hoisting rope. The hoisting machinery is supported e.g. on the vertical guide rails in the elevator shaft and so connected that its mass will assist the rope tensioning elements.
25

Fig. 2 presents a suspension arrangement that is better suited for a flat hoisting rope than the arrangement in
30 Fig. 1 because the hoisting rope does not intersect itself. The hoisting ropes are suspended in the same way as in the solution presented in Fig. 1. Each hoisting rope 5 is attached by its first end to the lower part of the elevator car 1, from where the ropes are passed to
35 the lower part of the counterweight 2 via the traction

sheave 7 of a drive machine 6 placed on the bottom of the elevator shaft below the elevator car 1 and via a diverting pulley 8 placed on the bottom of the elevator shaft below the counterweight. The hoisting ropes are
5 implemented in the same way as in Fig. 1, consisting of either a number of separate adjacent ropes or a single flat rope. The hoisting ropes descending from the elevator car go down to the traction sheave 7 by its back side as seen from the direction of the diverting pulley
10 8, pass around the traction sheave by its lower side and go further to the diverting pulley 8, pass around it by its lower side and go up to the counterweight. In this suspension model, however, the angle of contact between the hoisting rope and the traction sheave is substan-
15 tially smaller than in the solution presented in Fig. 1, in which it may be as large as over 270° . Therefore, the friction is also smaller, so the rope must be more tightly tensioned than in the case illustrated by Fig. 1. In other respects, the tensioning is implemented in
20 the same way as in Fig. 1.

Figures 3-6 present hoisting rope structures in which the load-bearing fibres are in strands. The strand layout is free and can be implemented either according to
25 load capacity requirements or according to bending capacity, e.g. torsional rigidity.

Fig. 3 presents a substantially flat elevator hoisting rope 5 as used in the suspension arrangement of the invention. It comprises six bundles 12a - 12e of strands
30 fitted in the same plane. The bundles consist of load-bearing strands 13. These longitudinal strands, which form the strength of the rope structure, are made of synthetic fibres, e.g. aramid fibres. The strands are
35 enclosed in a sheath 14 that binds the strands together

into a single structure and gives a good friction coefficient in contact with the traction sheave. The bundles 12a - 12f are fitted side by side to form a planar sheath 14, so that the width of the rope is considerably larger than its thickness. The sheath material 14 may be e.g. polyurethane, which gives a multifold friction coefficient as compared with a steel rope. If necessary, the planar surface of the sheath can be coated with various materials. The properties of the coating 15 regarding friction and wear can be optimised for different traction sheave materials. In Fig. 2, the bundles of strands are of a round shape in cross-section, but naturally, the shape can be chosen in accordance with the use.

15

Fig. 4 presents a flat hoisting rope solution in which the bundles 12 of strands are placed at different distances from each other. The Bundles are somewhat closer to each other near the edges than in the middle part of the hoisting rope. In the solution presented in Fig. 5, the bundles 12 of strands are placed non-symmetrically with respect to the longer midline of the hoisting rope, close to the friction surface of the rope. Fig. 6 presents a solution in which the strands and bundles 12 of strands of the hoisting rope are of different sizes in diameter. The larger bundles are placed at the edges of the rope as seen in its widthways direction, with smaller bundles placed between them. In the ways illustrated by Figures 4-6, it is possible to improve the tracking of the hoisting rope 5 as it is passing over the traction sheave or diverting pulleys.

Figures 7 and 8 present hoisting rope solutions in which the load-bearing fibres are in the form of a fabric. In the solution illustrated by Fig. 7, the fibres form in

the cross-section of the hoisting rope 5 lines crossing each other in both the longitudinal and lateral directions of the hoisting rope 5. The lines may also be in a position oblique to the longitudinal direction of the hoisting rope. Thus, the fabric may resemble e.g. the clinch-built, cross-ply structure of a car's safety belt or a corresponding belt. Fig. 8 presents a hoisting rope structure in which the hoisting rope in its entire cross-sectional area consists of fabric or fabrics bound together by a binding agent, e.g. polyurethane. By using different reinforcing fabrics, it is possible to produce a flexible hoisting rope or suspension rope in which the contacts between individual fibres can be increased or reduced as necessary.

15

The advantages achieved by using rope solutions as illustrated by Figures 3-8 include the following:

- When a single flat hoisting rope is used, the void space between ropes that is involved in the case of separate ropes is avoided, and thus the traction sheave can be made narrower than before.
- The cross-sectional area of the load-bearing part of the rope can be optimised.
- A good degree of damping of rope vibrations is achieved because the separate ropes are now replaced with bundles of strands embedded in a mass of vibration damping material.

30

When a thin, band-like hoisting rope is used, it is necessary to make sure that lateral drift of the hoisting rope off the traction sheave or diverting pulley is prevented. This can be done in various ways. In one solu-

35

tion, the traction sheave is provided with a tilting mechanism and sensors monitoring the position of the rope edge. The traction sheave is a straight cylinder, whose axis of rotation can be tilted to bring the hoisting rope to the central part of the traction sheave. When the hoisting rope is drifted to the edge of the traction sheave, a mechanical sensor or an equivalent detector based on beam of light or the like gives a corresponding signal to the system controlling the tilting of the traction sheave, whereupon the tilt of the traction sheave is altered so that the band-like hoisting rope is brought back to the middle of the traction sheave. If necessary, it is possible to use a cambered/crowned traction sheave or diverting pulley, i.e. one with a varying diameter, in which case the circumferential surface of the sheave/pulley is either convex or concave as seen from the front of the sheave/pulley. The advantage achieved is a good retention of the hoisting rope in its proper position.

20

When thin separate hoisting ropes are used, the bundles 12a - 12f of strands are placed apart from each other, in which case they function like independent hoisting ropes regardless of the other bundles.

25

As stated above, when the hoisting rope structure of the invention is used, the traction sheaves needed e.g. in the elevator suspension arrangements described above are considerably smaller in diameter and lighter than the traction sheaves currently used. The smaller traction sheave and machinery allow all elevator components to be accommodated in the elevator shaft, thus eliminating the need for a separate machine room. This brings considerable savings in the delivery price of the elevator.

35

It is obvious to a person skilled in the art that different embodiments of the invention are not restricted to the example described above, but that they may be varied in the scope of the claims presented below. Thus, 5 the elevator hoisting rope need not necessarily have a round or flat cross-sectional form. Instead, it may be e.g. a triangular-belt type rope having a V-shaped cross-section, in which case it is possible to achieve a very large friction between each hoisting rope and the 10 corresponding keyway on the traction sheave. The suspension ropes can also be made of synthetic fibres and they may consist of either several adjacent ropes or only one flat rope. In addition, the bundles of strands can be arranged in more than one layer, e.g. in two layers, if 15 necessary in view of the load to be borne by the rope. The suspension ratio may also be other than the 1:1 suspension presented in the example.

CLAIMS

1. Elevator rope arrangement for an elevator, in which an elevator car (1) and a counterweight (2) travelling
5 along guide rails in an elevator shaft are supported by suspension ropes (3), which are attached to the top part of the elevator car (1) and passed via at least one diverting pulley (4) to the counterweight (2), and in which at least one hoisting rope (5) is attached to the
10 elevator car (1) and passed from the elevator car to the counterweight (2) via the traction sheave (7) of a drive machine (6) and via at least one diverting pulley (8), **characterized** in that the hoisting rope (5) is a substantially thin rope made of synthetic material, with
15 rope tensioning arranged in the lower part of the elevator shaft.

2. Elevator arrangement as defined in claim 1, **characterized** in that the hoisting rope (5) is a substantially
20 thin rope made of synthetic fibres, such as aramid fibres, and having a sheath of plastic material, such as polyurethane.

3. Elevator arrangement as defined in claim 1 or 2,
25 **characterized** in that the hoisting rope (5) is a rope in which the bundles (12a-12f) of strands are made of synthetic fibres, e.g. aramid fibres, and the sheath (4) is made of plastic material, such as polyurethane, and that the bundles (12a-12f) have been fitted side by side in
30 at least one plane to form a layer of bundles of strands so that in cross-section the rope is substantially larger in width than in thickness.

4. Elevator arrangement as defined in claim 1 or 2,
35 **characterized** in that the hoisting rope (5) consists of

a number of adjacent ropes in which the bundles (12a-12f) of strands are placed separately from each other so that each bundle functions as an independent rope.

1/3

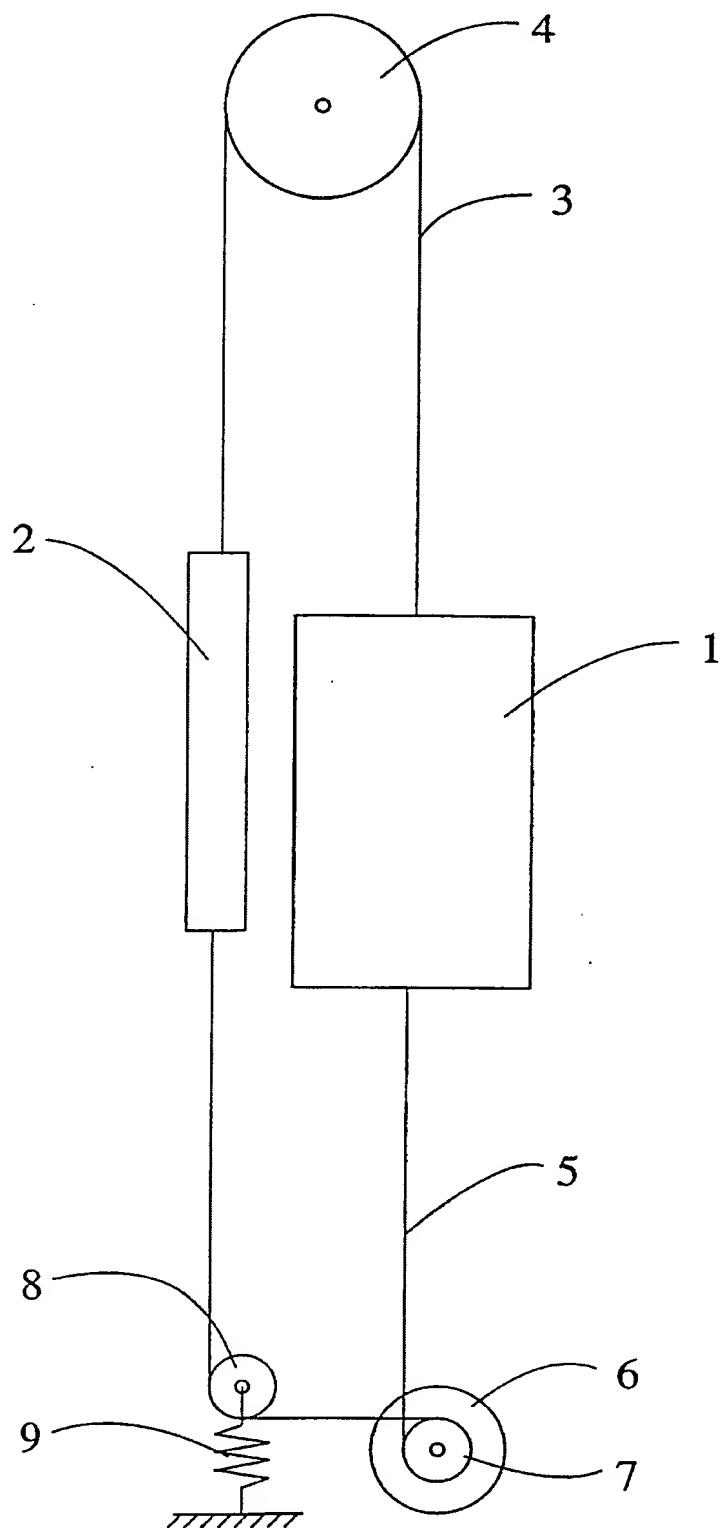
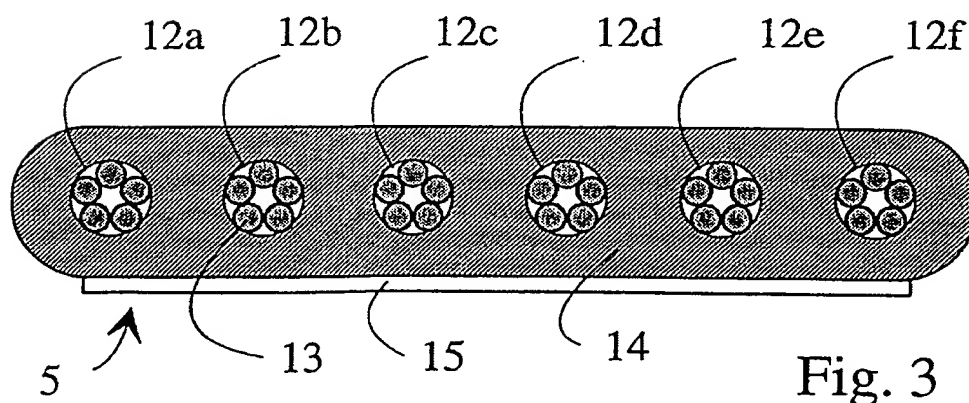
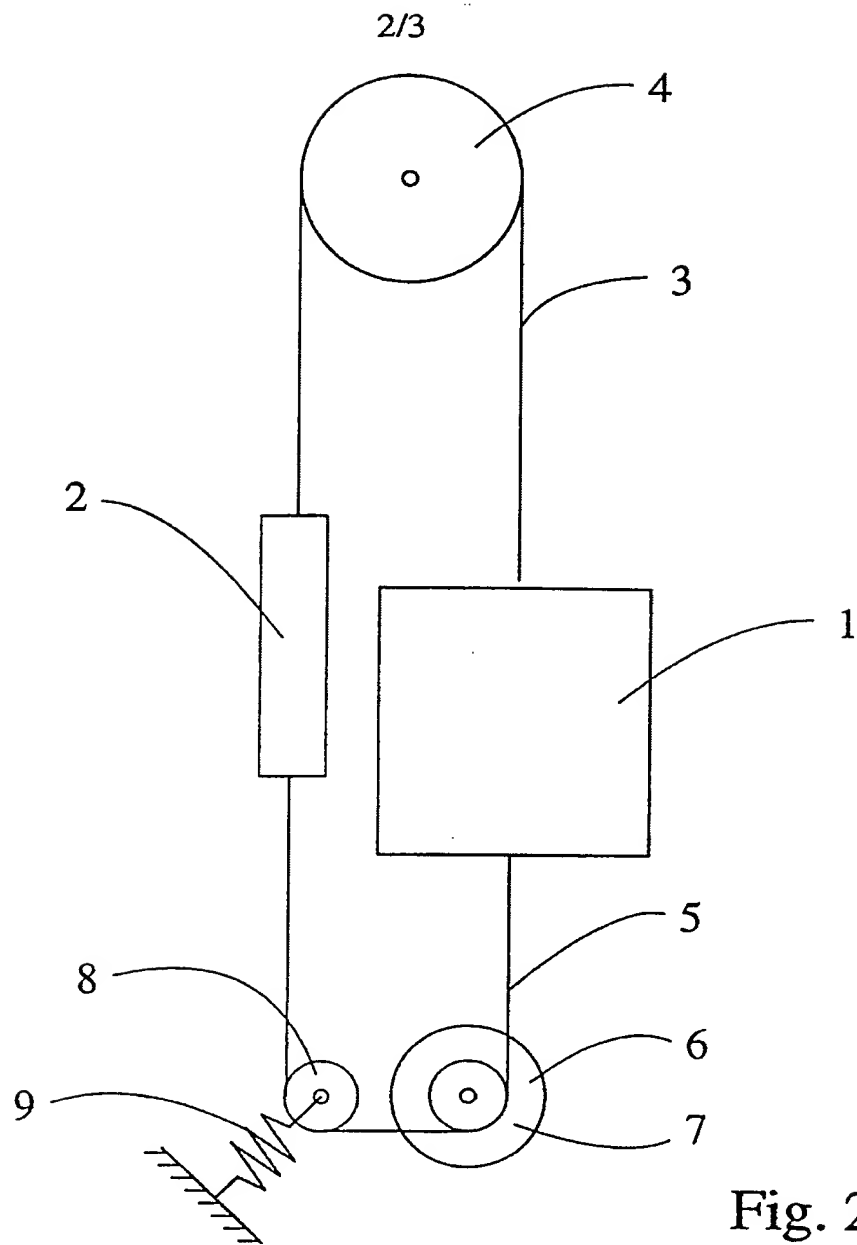
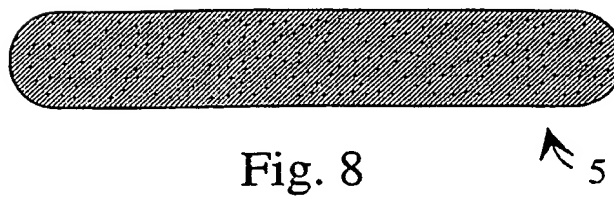
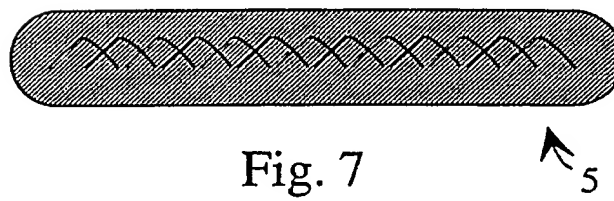
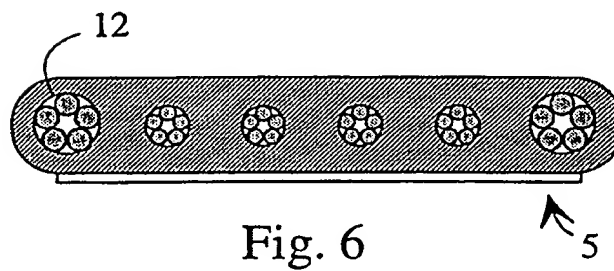
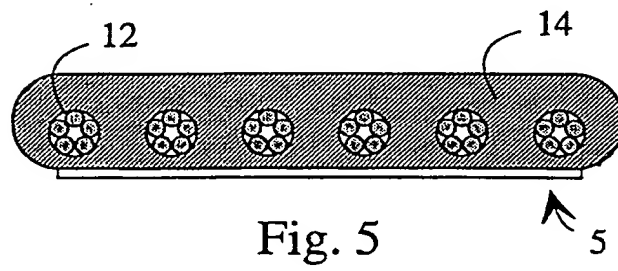
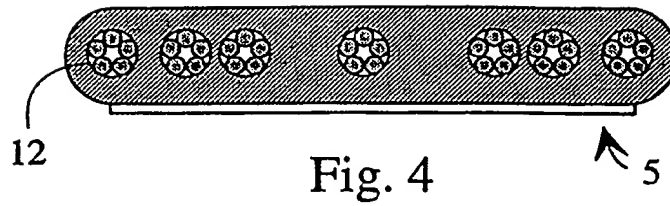


Fig. 1



3/3



INTERNATIONAL SEARCH REPORT

International application No.

PCT/FI 97/00824

A. CLASSIFICATION OF SUBJECT MATTER

IPC6: B66B 7/06, B66B 11/08

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

IPC6: B66B

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

SE,DK,FI,NO classes as above

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)

WPI, PAJ

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
Y	US 657380 A (C.W. & W.D. BALDWIN), 4 Sept 1900 (04.09.00), page 1, line 94 - page 2, line 2, figure 2 --	1-4
Y	US 5566783 A (K. YAMASHITA), 22 October 1996 (22.10.96), column 2, line 52 - line 67, figure 1 --	1-4
Y	EP 0672781 A1 (INVENTIO AG), 20 Sept 1995 (20.09.95) --	1-4
Y	US 4445593 A (J.D. COLEMAN ET AL), 1 May 1984 (01.05.84), column 3, line 10 - line 14; column 3, line 27 - line 39, figure 7 --	1-4

☒ Further documents are listed in the continuation of Box C.☒ See patent family annex.

* Special categories of cited documents:

"A" document defining the general state of the art which is not considered to be of particular relevance

"B" earlier document but published on or after the international filing date

"I" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)

"O" document referring to an oral disclosure, use, exhibition or other means

"P" document published prior to the international filing date but later than the priority date claimed

"T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention

"X" document of particular relevance: the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone

"Y" document of particular relevance: the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art

"&" document member of the same patent family

Date of the actual completion of the international search

2 April 1998

Date of mailing of the international search report

08.04.98

Name and mailing address of the ISA/

Swedish Patent Office

Box 5055, S-102 42 STOCKHOLM

Authorized officer

Mariana Eddin

INTERNATIONAL SEARCH REPORT

International application No.

PCT/FI 97/00824

C (Continuation). DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
A	US 5398781 A (J. BIEWALD ET AL), 21 March 1995 (21.03.95), figure 1 --	1
A	US 3174585 A (D.D. TOFANELLI), 23 March 1965 (23.03.65) -- -----	1

INTERNATIONAL SEARCH REPORT

Information on patent family members

02/03/98

International application No.

PCT/FI 97/00824

Patent document cited in search report			Publication date	Patent family member(s)		Publication date
US	657380	A	04/09/00	NONE		
US	5566783	A	22/10/96	EP	0669276 A	30/08/95
				JP	2536816 B	25/09/96
				JP	7237861 A	12/09/95
EP	0672781	A1	20/09/95	AU	682743 B	16/10/97
				AU	1353495 A	07/09/95
				BR	9500779 A	24/10/95
				CA	2142072 A	03/09/95
				CZ	282660 B	13/08/97
				CZ	9500523 A	12/03/97
				DE	59403165 D	00/00/00
				EP	0639248 A,B	22/02/95
				FI	950936 A	03/09/95
				JP	7267534 A	17/10/95
				JP	8500657 T	23/01/96
				NO	950796 A	04/09/95
				NZ	270477 A	28/10/96
				PL	307384 A	04/09/95
				US	5526552 A	18/06/96
				US	5566786 A	22/10/96
				ZA	9501692 A	08/12/95
				CN	1121040 A	24/04/96
				HU	70630 A	30/10/95
				HU	9500433 D	00/00/00
US	4445593	A	01/05/84	NONE		
US	5398781	A	21/03/95	AT	129984 T	15/11/95
				DE	9201374 U	02/04/92
				DE	59300869 D	00/00/00
				EP	0554712 A,B	11/08/93
				ES	2082528 T	16/03/96
				JP	2041104 C	09/04/96
				JP	6080347 A	22/03/94
				JP	7076074 B	16/08/95
				US	5437347 A	01/08/95
US	3174585	A	23/03/65	NONE		